



# Hydraulic Pumps T6CM for low pressure drives

Denison Vane Technology



ENGINEERING YOUR SUCCESS.

**Main Technical Data**

**T6CM, Denison Vane Pumps**

**Key features**

These hydraulic vane pumps are using the Denison 3 springs technology which has been proven to be very well adapted to pumping at very low system pressure even fluids with high viscosity.

**Reliable performance**

The Denison Vane Technology allows very low noise levels over the entire operating range and during the whole life of the pump. Like all our vane pumps, the performances do remain very stable over time, making this pump an ideal solution for the heavy duty industries.

**Long lifetime**

The fully pressure balanced concept increases the pump lifetime over its full operating range while double lip vanes reduce the sensitivity to fluid pollution.

**General characteristics**

	Mounting standard	Weight without connector and bracket - kg	Moment of inertia kgm <sup>2</sup> x 10 <sup>-4</sup>	SAE 4 bolts J518c - ISO/DIS 6162-1	
				Suction	Pressure
T6CM T6CMY	SAE J744c ISO/3019-1 SAE B	15,7	7,5	1.1/2"	1"

**Main technical data**

Pump		Theoretical Displacement Vi cm <sup>3</sup> /rev.	Max. Speed Mineral oil rpm	Max. Pressure		Minimum allowable inlet pressure (bar absolute)					Ring size				
Type	Ring size			Int. bar	Cont. bar	Speed rpm									
						1200	1500	1800	2100	2200					
T6CM T6CMY	R03	10,8	2200	140	110	0,80	0,80	0,80	0,80	0,80	R03				
	R05	17,2										0,80			
	R06	21,3													
	R08	26,4													
	R10	34,1													
	R12	37,1													
	R14	46,0													
	R17	58,3													
	R20	63,8													
	R22	70,3											0,85	0,90	R22
	R25	79,3											0,90	0,95	R25
	R28	88,8											0,90	0,98	R28
R31	100,0	90	75	0,85	0,90	R31									

Inlet pressure is measured at inlet flange with petroleum base fluids at viscosity between 10 and 65 cSt. The difference between inlet pressure at the pump flange and atmospheric pressure must not exceed 0.2 bar to prevent aeration.

Multiply absolute pressure by 1,25 for HF-3, HF-4 fluids.  
by 1,35 for HF-5 fluid.  
by 1,10 for ester or rapeseed base.

Note : For further information or if the performance characteristics outlined in the table do not meet your particular requirements, please consult your local Parker office.



# Ordering Code & Dimensions

# Hydraulic Pumps for low pressure drives T6CM, Denison Vane Pumps

## Model No.

**T6CM (Y) - R22 - 1 R 00 - C 1 - ..**

### T6CM series - SAE B 2 bolts

J744 mounting flange

Y = Port flanges with metric threads

### Displacement

Volumetric displacement (cm<sup>3</sup>/rev.)

R03 = 10,8

R17 = 58,3

R05 = 17,2

**R20 = 63,8**

R06 = 21,3

R22 = 70,3

R08 = 26,4

R25 = 79,3

R10 = 34,1

R28 = 88,8

R12 = 37,1

R31 = 100,0

R14 = 46,0

### Type of shaft

**1 = keyed (SAE B)**

2 = keyed (non SAE)

3 = splined (SAE B) 13 teeth

4 = splined (SAE BB) 15 teeth

### Modifications

#### Seal class

**1 = S1 (for mineral oil)**

4 = S4 (for the resistant fluids)

5 = S5 (for mineral oil and fire resistant fluids)

#### Design letter

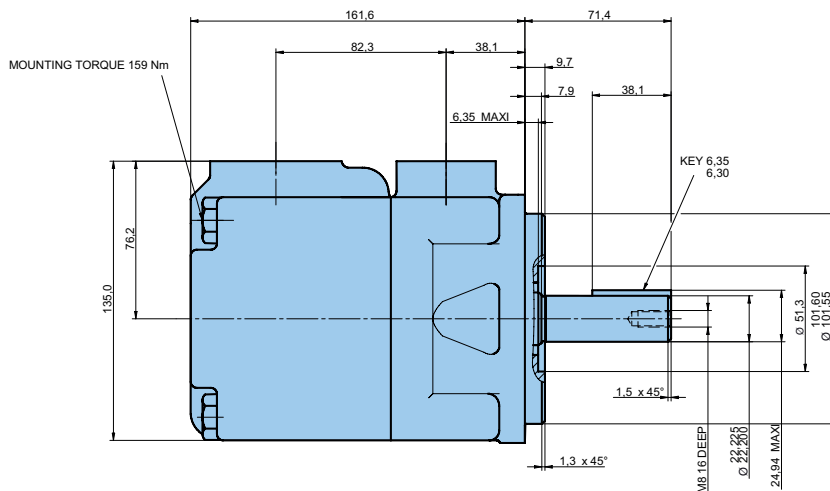
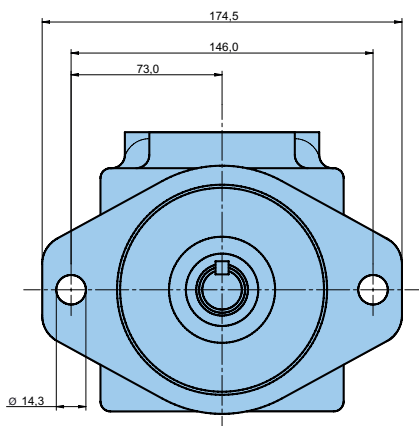
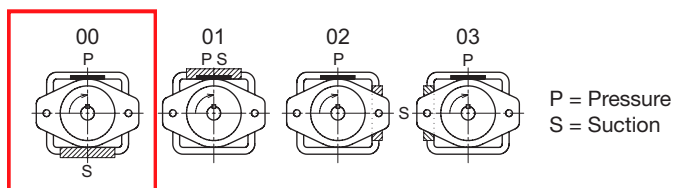
#### Porting combination

**00 = standard**

#### Direction of rotation (shaft end view)

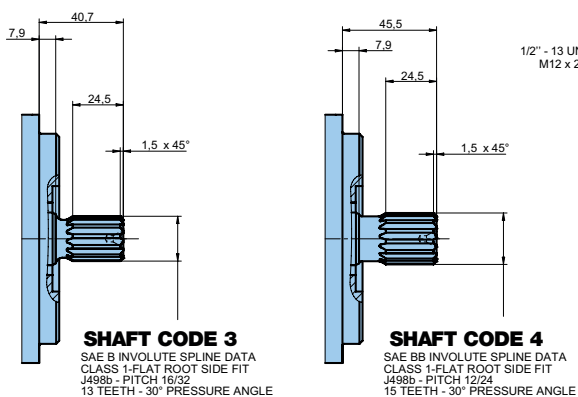
**R = Clockwise**

L = Counter-clockwise



### SHAFT CODE 1

(KEYED SAE B)

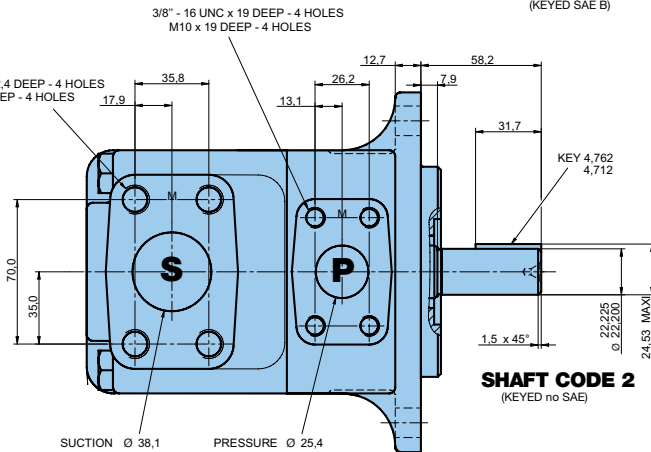


### SHAFT CODE 3

SAE B INVOLUTE SPLINE DATA  
CLASS 1-FLAT ROOT SIDE FIT  
J498b - PITCH 16/32  
13 TEETH - 30° PRESSURE ANGLE

### SHAFT CODE 4

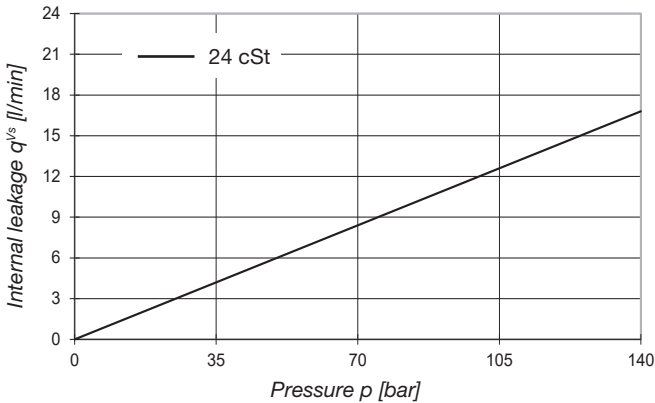
SAE BB INVOLUTE SPLINE DATA  
CLASS 1-FLAT ROOT SIDE FIT  
J498b - PITCH 12/24  
15 TEETH - 30° PRESSURE ANGLE



### SHAFT CODE 2

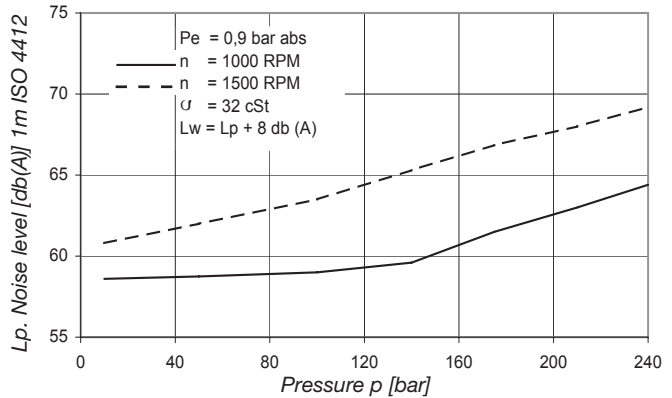
(KEYED no SAE)

**Internal leakage (Typical)**

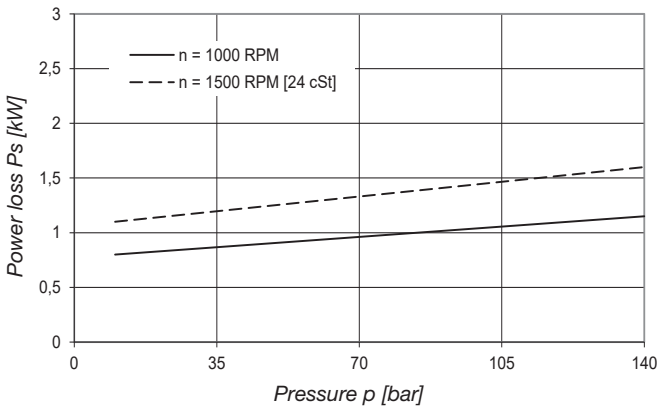


Do not operate pump more than 5 seconds at any speed or viscosity if internal leakage is higher than 50% of theoretical flow.

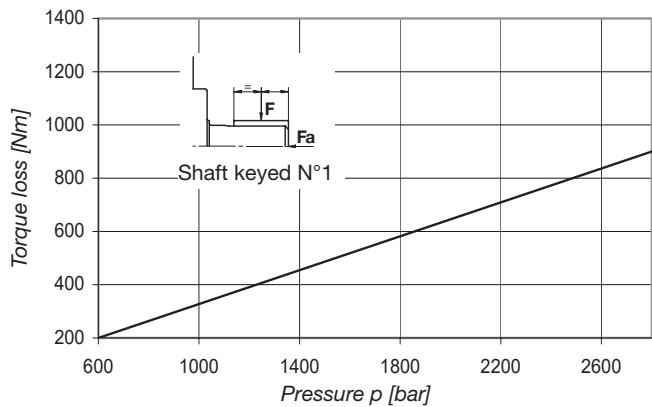
**Noise level (Typical)  
T6CM - R22**



**Power loss hydromechanical (Typical)**



**Permissible radial load**



Max permissible axial load Fa = 800 N

**Operating characteristics - typical [24 cst]**

Pressure port	Series	Vi Volumetric displacement	Output flow q <sub>v</sub> [l/min] @ 1500 RPM			Input power P [kW] @ 1500 RPM		
			p = 0 bar	p = 50 bar	p = 140 bar	p = 7 bar	p = 50 bar	p = 140 bar
T6CM T6CMY	R03	10,8 cm <sup>3</sup> /rev	16,2	10,2	<sup>1)</sup>	1,3	2,7	<sup>1)</sup>
	R05	17,2 cm <sup>3</sup> /rev	25,8	19,8	<sup>1)</sup>	1,4	3,0	<sup>1)</sup>
	R06	21,3 cm <sup>3</sup> /rev	31,9	25,9	<sup>1)</sup>	1,5	4,0	<sup>1)</sup>
	R08	26,4 cm <sup>3</sup> /rev	39,6	33,6	22,8	1,6	4,6	11,0
	R10	34,1 cm <sup>3</sup> /rev	51,1	45,1	28,3	1,7	5,6	13,7
	R12	37,1 cm <sup>3</sup> /rev	55,6	49,6	32,8	1,8	6,0	14,7
	R14	46,0 cm <sup>3</sup> /rev	69,0	63,0	52,2	2,0	7,1	17,9
	R17	58,3 cm <sup>3</sup> /rev	87,4	81,5	70,7	2,2	8,6	22,2
	R20	63,8 cm <sup>3</sup> /rev	95,7	89,7	78,9	2,3	9,3	24,1
	R22	70,3 cm <sup>3</sup> /rev	105,4	99,5	88,7	2,4	10,1	26,1
	R25	79,3 cm <sup>3</sup> /rev	118,9	113,0	102,2	2,5	11,2	29,6
	R28	88,8 cm <sup>3</sup> /rev	133,2	127,2	116,4	2,8	12,4	32,9
	R31	100,0 cm <sup>3</sup> /rev	150,0	144	139,2	2,9	13,8	<sup>2)</sup>

<sup>1)</sup> Do not use : η vol < 50 %

<sup>2)</sup> p max = 90 bar

**Pump Selection**

**Calculation**

To resolve

Volumetric displ. \_\_\_\_\_  $V_i$  [cm<sup>3</sup>/rev.]  
 Available flow \_\_\_\_\_  $Q_{eff}$  [l/min]  
 Input power \_\_\_\_\_  $P_{eff}$  [kW]

Performances required

Requested flow \_\_\_\_\_  $Q$  [l/min] 42  
 Speed \_\_\_\_\_  $n$  [rpm] 1500  
 Pressure \_\_\_\_\_  $p$  [bar] 50

Routine :

1. First calculation  $V_i = 1000 \times Q / n$

2. Choose the pump with the next higher  $V_i$

3. Theoretical flow of this pump

$$Q_{theo} = V_i \times n / 1000$$

4. Read  $q_{vs}$  leakage function of pressure  $Q_{vs} = f(p)$  on curve

5. Available flow  $Q_{eff} = Q_{theo} - q_{vs}$

6. Theoretical input power

$$P_{theo} = Q_{theo} \times p / 600$$

7. Read  $P_s$  hydrodynamical power loss on curve

8. Calculation of necessary input power

$$P_{eff} = P_{theo} + P_s$$

9. Results

Example :

$$V_i = 1000 \times 42 / 1500 = 28 \text{ cm}^3/\text{rev.}$$

$$R10, V_i = 34,1 \text{ cm}^3/\text{rev.}$$

$$Q_{theo} = 34,1 \times 1500 / 1000 = 51,1 \text{ l/min}$$

$$q_{vs} = 6 \text{ l/min at 50 bar}$$

$$Q_{eff} = 51,1 - 6 = 45,1 \text{ l/min}$$

$$P_{theo} = 51,1 \times 50 / 600 = 4,3 \text{ kW}$$

$$P_s \text{ at 1500 rpm, 50 bar} = 1,3 \text{ kW}$$

$$P_{eff} = 4,3 + 1,3 = 5,6 \text{ kW}$$

$$V_i = 34,1 \text{ cm}^3/\text{rev.}$$

$$Q_{seff} = 45,1 \text{ l/min}$$

$$P_{eff} = 5,6 \text{ kW}$$

T6CM(Y) R10

Follow these calculation steps for each application.

**General applications instructions**

1. Check speed range, pressure, temperature, fluid quality, fluid viscosity and pump rotation way.
2. Check the inlet conditions of the pump, if it can accept the application requirements.
3. Check the type of shaft : if it will support the operating torque.
4. Check the coupling which must be chosen to minimize pump shaft load (weight, misalignment).
5. Filtration : must be adequate for the lowest contamination level.
6. Check the environment of the pump : as to avoid noise reflection, pollution and shocks.



**WARNING – USER RESPONSIBILITY**

**FAILURE OR IMPROPER SELECTION OR IMPROPER USE OF THE PRODUCTS DESCRIBED HEREIN OR RELATED ITEMS CAN CAUSE DEATH, PERSONAL INJURY AND PROPERTY DAMAGE.**

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